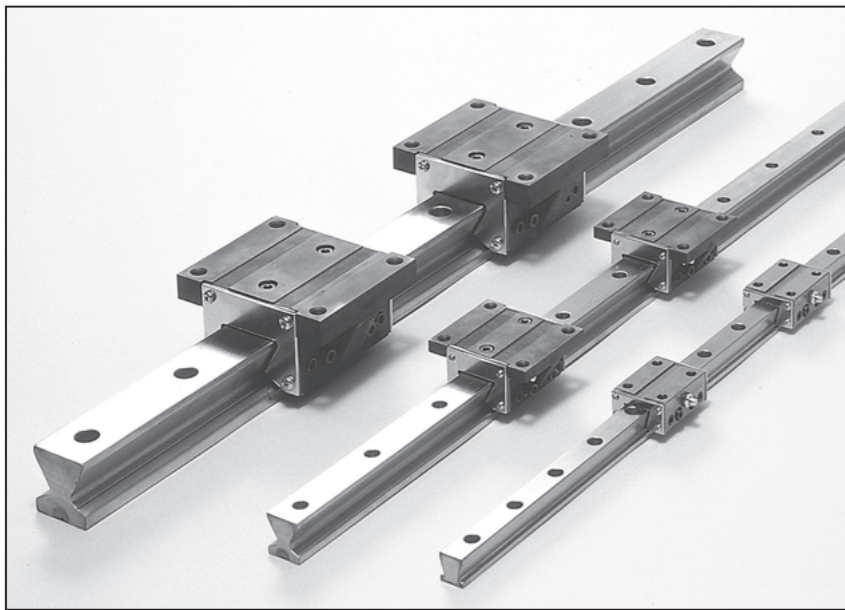


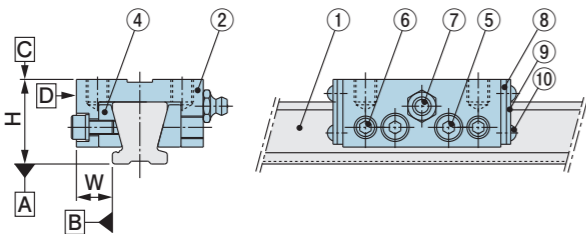
# Oiles Slide Shifter S Type **STC/STF GR**

RoHS2 ELV

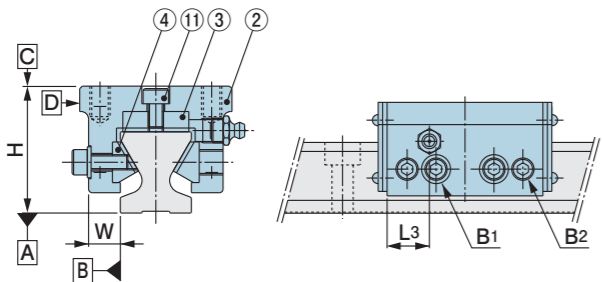


## Component Parts · Accuracy

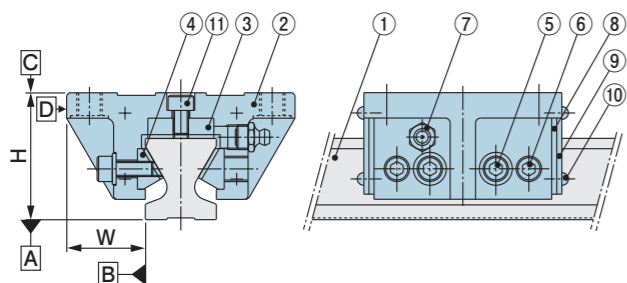
### STC20



### STC28



### STF28,38,48 Flange type



## Component Parts

No.	Name	Material
①	Guide rail	S45C+hard chrome-plated
②	Shift table	FCD450
③	Liner (Sliding material)	Oiles metal
④	Gib (Sliding material)	Oiles metal
⑤	Adjust pull bolt	Hex socket head cap bolt
⑥	Adjust push bolt	Hex socket head cap bolt
⑦	Grease nipple	A-PF1/8 (Screw mounting hole size Rp 1/8)
⑧	Seal	Polyurethane
⑨	Seal fixing plate	SPCC + rust proof
⑩	fixing screws	—
⑪	Liner fixing bolts	—

※About the shift table  
 · STC20 has Oiles metal joined to FCD450.  
 · STC28, STF28, 38 and 48 have been treated with a zinc phosphate coating for rustproofing (dyed black).

## Accuracy

(Unit: mm)

Items	Accuracy
Running parallelism of side C with respect to side A	※0.03 or less/m (upright)
Running parallelism of side D with respect to side B	※0.05 or less/m (upright)
Tolerance of H dimension	0 -0.1
Tolerance of W dimension	-0.1 -0.3

※These are values obtained after installing and correcting the guide rails.

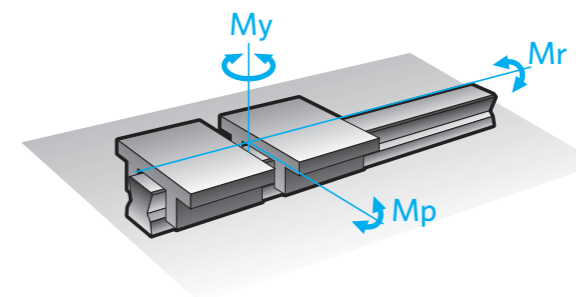
## Service Range

### Allowable load

- Static allowable load: Allowable load when a load is applied at a stationary condition or at quite low speed near stopping (not more than 0.0017 m/s [0.1 m/min.])
- Dynamic allowable load: Allowable load in the condition with sliding speed of 1.0 m/s [60 m/min] or less.
- Do not use STC20 in lateral or hanging position.

### Allowable Moment

- 2 tables on 1 rail



Part No.	Type	Allowable load N [kgf]			Cases	Allowable moment N · m [kgf · m]		
		Table position				Moment direction (when contact with the table)		
		Upright	Lateral	Hanging		Mp	Mr	My
STC20	Static	10,800 {1,100}	—	—	1 rail 2 tables	49 { 5}	15 { 1.5}	44 { 4.5}
	Dynamic	3,430 { 350}	—	—				
STC28 STF28	Static	17,700 {1,800}	4,410 { 450}	3,920 { 400}	1 rail 2 tables	200 { 20}	20 { 2.0}	180 { 18}
	Dynamic	5,880 { 600}	1,470 { 150}	1,270 { 130}				
STF38	Static	31,400 {3,200}	7,850 { 800}	7,360 { 750}	1 rail 2 tables	320 { 33}	30 { 3.0}	290 { 30}
	Dynamic	10,800 {1,100}	2,650 { 270}	2,450 { 250}				
STF48	Static	44,100 {4,500}	13,700 {1,400}	11,800 {1,200}	1 rail 2 tables	490 { 50}	40 { 4.0}	440 { 45}
	Dynamic	14,700 {1,500}	4,410 { 450}	3,920 { 400}				

※The value of moment direction when 2 table in 1 rail is a reference value.  
 ※Use the formula on page 337 to calculate the allowable load and determine whether it can be used.

## Allowable Velocity

Lubrication conditions	Allowable max velocity	Remarks
Dry	0.5m/s {30m/min}	—
Periodical Inbrication	1.0m/s {60m/min}	Apply lubrication every 10 km of sliding

※Greasing is needed if the stroke is 1 meter or more or the allowable wear amount is small.

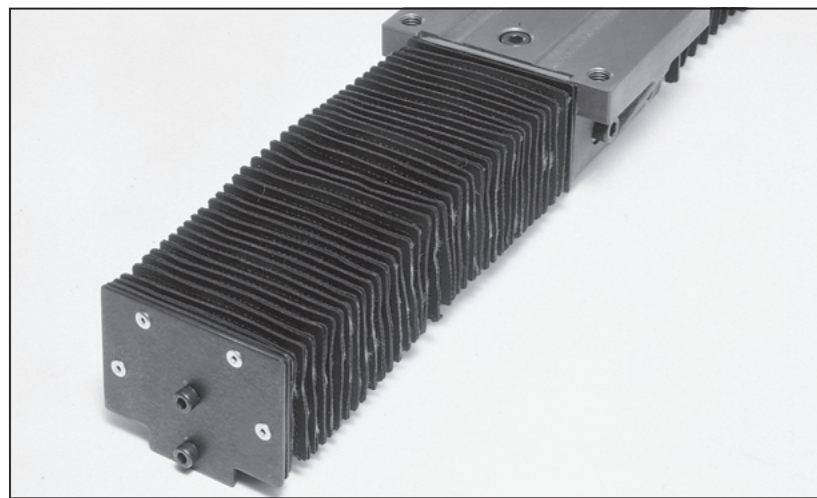
## Seal Friction Fs

Part No.	STC20	STC28	STF28	STF38	STF48
Fs	9.8N {1.0kgf}	12N {1.2kgf}	12N {1.2kgf}	15N {1.5kgf}	18N {1.8kgf}





# Optional Parts Applicable to S and SE Types



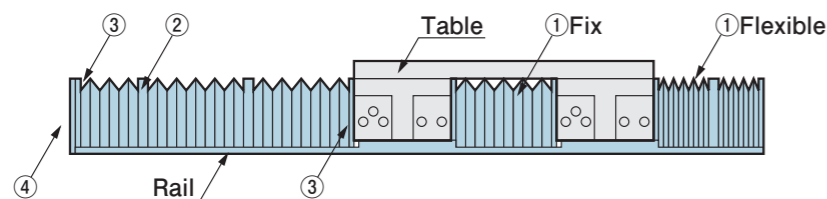
## Bellows exclusive for slide shifters

The S and SE types incorporate Oiles bearings on the sliding surfaces and have superior foreign matter resistance. It is recommended to use the exclusive bellows if higher resistance is required. A heat-resistant bellows is also available.

## Caps for covering up bolt holes on guide rails

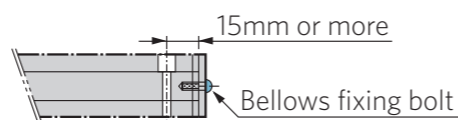
Exclusive caps for preventing dust, etc. from entering the bolt holes for mounting the guide rail are available.

## Component Parts for Bellows Exclusive for Slide Shifters



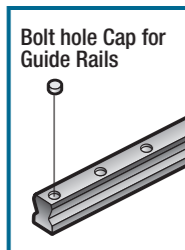
No.	Name	Material
①	Flexible bellows	Neoprene rubber + nylon cloth
	Fix bellows	
	Heat-resistant flexible bellows	
	Heat-resistant fix bellows	
②	Slide Plate	PVC
③	Clamp Plate	SPCC
④	End Plate	SPCC

※ Determine the positions of the bolts for fixing the rail at both ends when using the 38J or 48J as shown below.



## Bolt hole Cap for Guide Rails

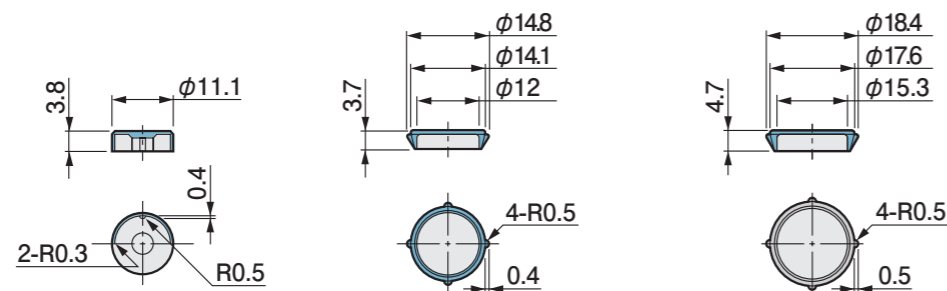
Exclusive bolt hole cap (plastic) is available to keep out the dust from bolt holes.



■ CP-6 (for M6)

■ CP-8 (for M8)

■ CP-10 (for M10)



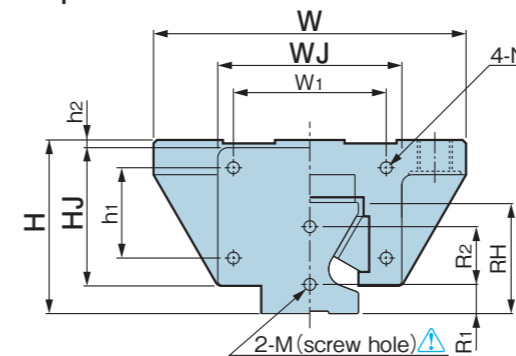
Part No.	Bolt Size	Rails
CP-6	M6	GR20、28、GRE20、28
CP-8	M8	GR38
CP-10	M10	GR48

※ Press fit the cap with a plastic hammer.  
 ※ Fit CP-6 in the clearance between the rail and the bolt, and twist it to tie up.

## Product Identification for Exclusive Bellows for Slide Shifters

### CAUTION

■ Mounting screw hole on the guide rail for bellows is optional.



■ Flexible Bellows

Part No. **28J - A - 100 - T**

Put T for heat-resistant bellows  
 Shortened length of bellows (Lmin)  
 Bellows expansion ratio A or B

■ Fix Bellows

Part No. **28JK - 140 - T**

Put T for heat-resistant bellows  
 Length of fixed bellows

● End plate of 20J sticks out 8mm from the table surface.

Part No.	W×H	Bellows size WJ×HJ	Expansion ratio A	Stroke	Expansion ratio B	Stroke	h1	h2	W1	RH	R1	R2	N	M	Applicable tables
20J	45×30	52×32	5	under 1100	3.5	1100 or more	14	8	37	22	6	10	M3×10	M4×8	STC20 STE20
28J	90×50	60×40	5	under 1100	3.5	1100 or more	26	1	44	32	8	18	M3×10	M4×8	STC28 STF28 STE28 STFE28
38J	110×65	80×52	7	under 1300	5.5	1300 or more	36	1	58	42	10	24	M4×12	M5×10	STF38
48J	140×82	101×67	10	under 1300	7.5	1300 or more	50	3	74	52	12	30	M6×12	M6×10	STF48

## Calculating formula

■ Length of Bellows

(L min=Shortened length, L max=Expanded length)

In case of expansion ratio A

$$L \text{ min} = \frac{S}{A-1}, L \text{ max} = L \text{ min} \times A$$

In case of expansion ratio B

$$L \text{ min} = \frac{S}{B-1}, L \text{ max} = L \text{ min} \times B$$

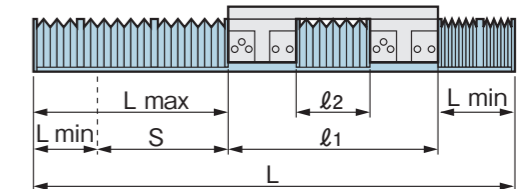
■ Total length of guide rail when using bellows

Using bellows at both ends  
 $L = (L \text{ min} \times 2) + S + l_1$

Using bellows at one end  
 $L = L \text{ min} + S + l_1$

In case of standard guid rails, L min dimension needs to be adjusted.

$$L \text{ min} = \frac{L - S - l_1}{2}$$



S: Stroke  
 A, B: Expansion ratio of bellows  
 L max: Expanded length of bellows  
 L min: Shortened length of bellows  
 l1: Table length  
 l2: Fix bellows length  
 L: Rail length

## Calculation example

Expression of calculating the bellows length is  $L \text{ min} = \frac{S}{A-1}$

$$L \text{ min} = \frac{400}{5-1} = 100\text{mm}$$

Required rail length  $L_1 = (L \text{ min} \times 2) + S + l_1$

$$L_1 = (100 \times 2) + 400 + 300 = 900\text{mm}$$

Bellows length L min when using standard rail length L2 (1000mm)

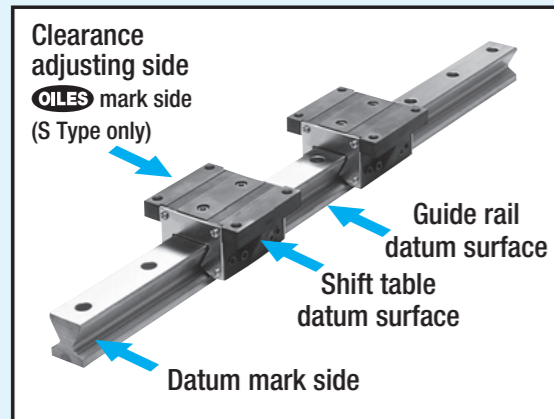
$$L \text{ min} = (1000 - 400 - 300) / 2 = 150\text{mm}$$

STF28 Stroke: S=400mm  
 Expansion ratio: A=5  
 Table length: l1=300mm  
 Fix bellows length: l2=140mm  
 Required rail length: L1  
 Standard rail length: L2=1000mm

# Installation and Adjusting Methods of S and SE Types

- The clearance between the shift table and guide rail of the S type need be adjusted.
- The shift table of the SE type has an automatic clearance adjustment function.

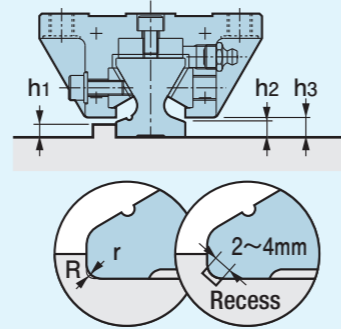
## Installation Datum Surface



The guide rail and shift table have their own datum surface for correct installation. The datum surface of the guide rail is the datum mark side (side B). That of the shift table is the opposite side (side D) to the OILES mark.

※Side B and D refer to the datum mark on page 287.

## Stage machining of installation datum part and corner dimensions

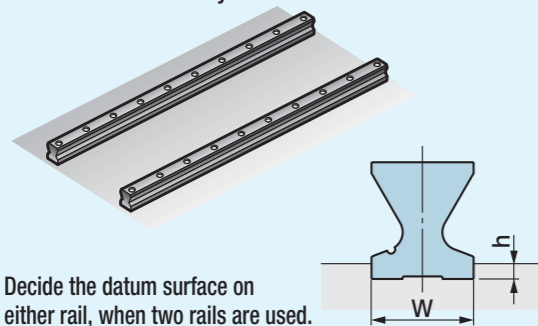


(Unit: mm)

Part No.	h1	h2	h3	r	R mating corner
GR20-GRE20	3~4	4	6	R1	R0.5 or less or recess
GR28-GRE28	4~6	6	8	R1.5	R1 or less or recess
GR38	5~8	8	10	R1.5	Ditto
GR48	5~8	10	11	R2	R1.5 or less or recess

## Installation of Guide Rails

It is recommended that the guide rail be corrected before installation. The product alone has a bend of not more than 0.2 mm/m, which also applies to both the S and SE types. When it is installed on a base, the bend is corrected below 0.03 mm/m. After correction, adjust bend of the S type rail by means of clearance adjustment of the shift table. The SE type has an automatic clearance adjustment function and adjusts clearance automatically.

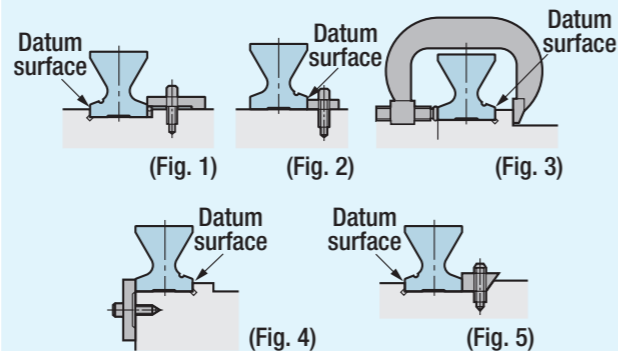


Decide the datum surface on either rail, when two rails are used.

Part No.	W	h
GR20 · GRE20	19.5 <sup>+0.08</sup> / <sub>+0.05</sub>	2~3
GR28 · GRE28	28 <sup>+0.08</sup> / <sub>+0.05</sub>	3~4
GR38	38 <sup>+0.08</sup> / <sub>+0.05</sub>	4~5
GR48	48 <sup>+0.08</sup> / <sub>+0.05</sub>	4~5

## Guide Rail Installation Adjustment Example

- ① Make a groove along the guide rail axis. Press the rail against the datum surface strongly to correct it. When two rails are used, parallelism is secured easily if grooves are made simultaneously.
- ② Alternative procedures are as shown below: Make the widths of the installation grooves roughly, insert drill rods and rails into the grooves, and fix the rails while pressing the drill rods. (See Fig. 1.)
- ③ Other procedures as shown below: Install a rigid plate on a planar base, and install the rail to fit this plate. (See Fig. 2.)
- ④ Make stages on the mating base with a planer or milling machine, press the datum surface of the rail against the machined surface with a vice or bolts and auxiliary plate, and fix the rail. (See Figs. 3, 4 and 5.)

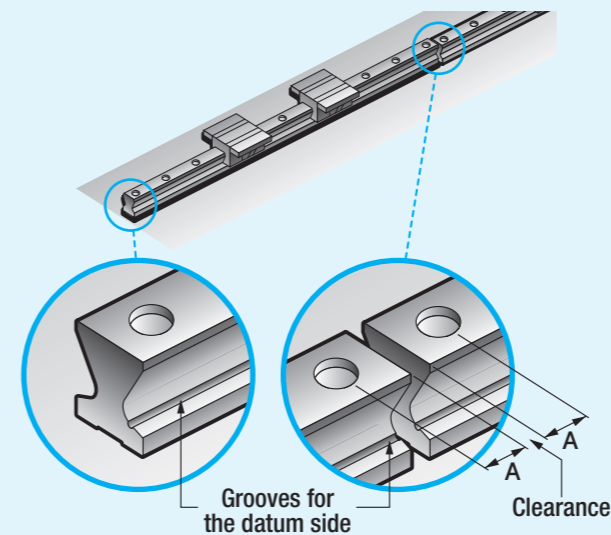


## Connecting Guide Rails

### Joining guide rails

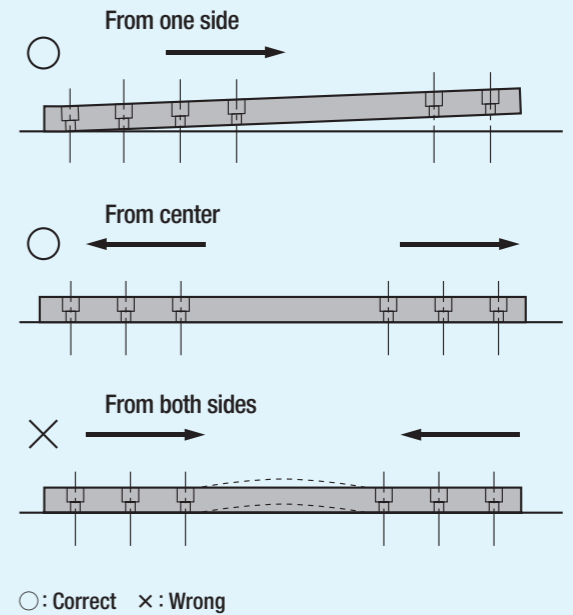
Fix the guide rails with the grooves with the datum marks on the same side.

The distance A between the mounting hole and end face is machined with a minus tolerance and accordingly the joint has a clearance. However, the clearance causes no bad influences.

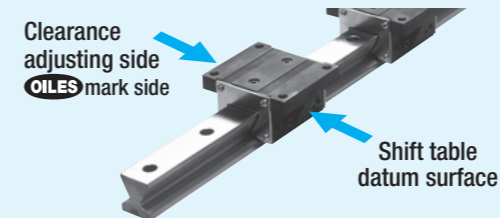


## Cautions

Tighten the bolts of the rail from one side or from the center to the right and left in sequence.

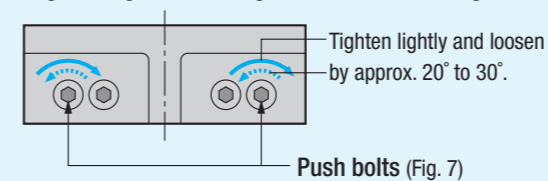


## Clearance Adjusting Method (S Type)

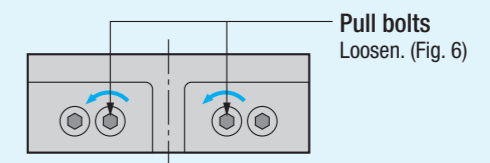


Recommended tightening torque	
STC20	1.47N · m {15kgf · cm}
STC28 · STF28	1.96N · m {20kgf · cm}
STF38	2.45N · m {25kgf · cm}
STF48	2.94N · m {30kgf · cm}

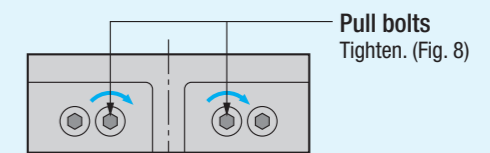
- ② Tighten the push bolts lightly, check the clearance zero condition, press reversely by approximately 20 to 30°, and return the bolts. (See Fig. 7.) For fine adjustment, retry adjustment in the order of Fig. 7 and Fig. 8. Clearance is increased or decreased by adjusting the degree of losing the bolts shown in Fig. 7.



- ① Loosen the pull bolts on the side with OILES mark (on the grease nipple side) sufficiently. (See Fig. 6.)



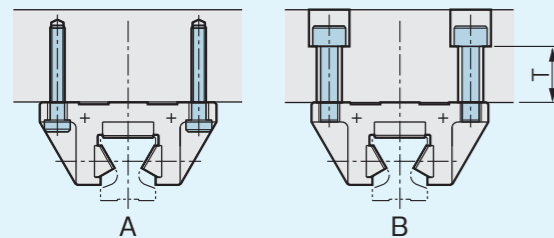
- ③ Tighten the pull bolts. Clearance becomes 0.03 to 0.05 mm. (Fig. 8.) NOTE: Tighten the pull bolts until the spring washers collapse. However, do not tighten them more strongly.



# Installation and Adjusting Methods of S and SE Types

## Installation of Shift Tables

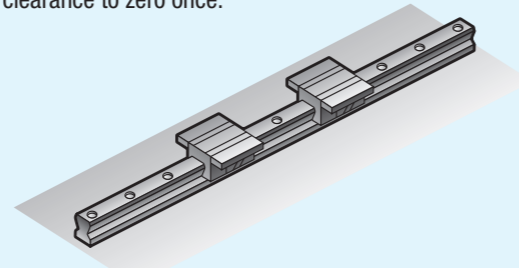
The shift table fixing bolts may be used in two ways as shown below. The recommended bolt diameters and lengths are as shown below.



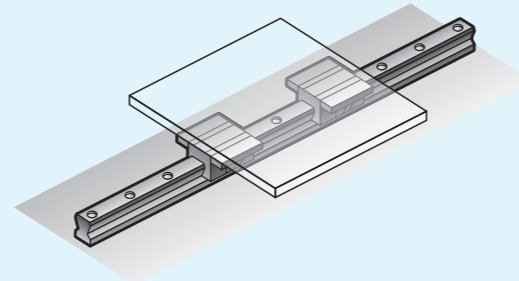
Part No.	A	B
STE20 · STC20	—	M8×(T+ 5)
STE28 · STC28	—	M8×(T+ 8)
STFE28 · STF28	M8×20	M10×(T+ 8)
STF38	M8×25	M10×(T+12)
STF48	M10×30	M12×(T+16)

## Installing Several Shift Tables on a Single-axis Rail (S Type)

① Insert the shift tables in the guide rail and adjust the clearance to zero once.

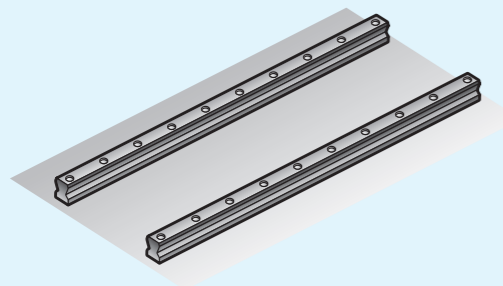


② Fasten the mating plate to the shift tables finally, adjust the linear accuracy, and adjust the clearance.

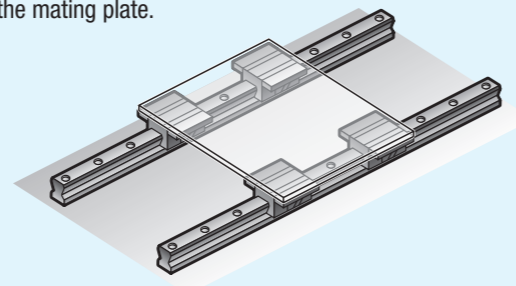


## Installing Several Shift Tables on Dual-axis Rails (S Type)

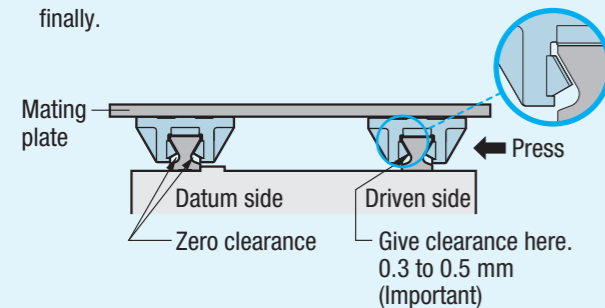
① Make sure that two rails are in parallel. (0.2 mm or less)



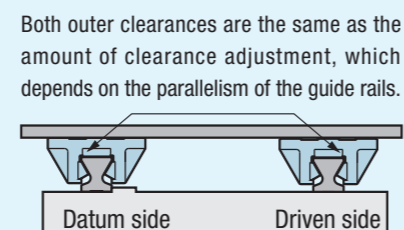
② Insert the shift tables into the guide rails with the clearance adjusting sides (OILES marks) outward. Put the mating plate.



③ Zero the clearance of the datum-side shift tables. Set the clearance of the driven shift tables to 0.3 to 0.5 mm, press the shift tables against the rails in the direction of the arrow, and fix the shift tables to the mating plate finally.

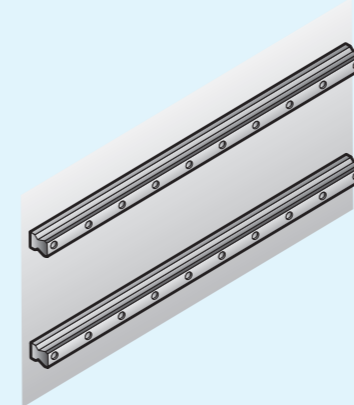


④ Adjust the clearance on the datum-side shift tables.

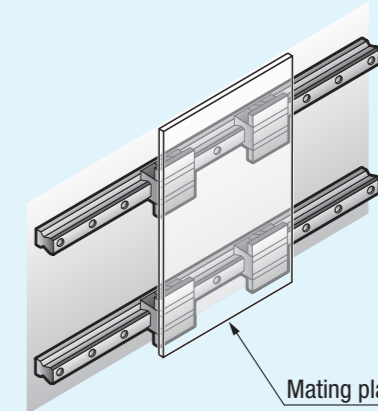


## Installing Several Shift Tables on Dual-axis Rails Topsy-turvy, Laterally, or Vertically (S Type)

① Make sure that two rails are in parallel. (0.2 mm or less)



② Insert the shift tables with adjusted clearance into the guide rails.



③ Fix the mating plate to the shift tables temporarily, make sure that the tables move smoothly, and fix the mating plate finally.

④ Recheck parallelism and clearance of the rails if movement is not smooth. If large moment loads are applied, the resistance increases.

## Other Instructions

① Use knock pins for both the guide rails and shift tables if vibrations or large impact loads are applied to them. The fixing holes of the guide rails may be used at intervals of several holes for the knock pins.

② It is recommended that a mating plate with high parallelism should be used. If sufficient parallelism cannot be secured for reasons of machining, carry out adjustment with shims so that the guide rails and shift table are in good contact.

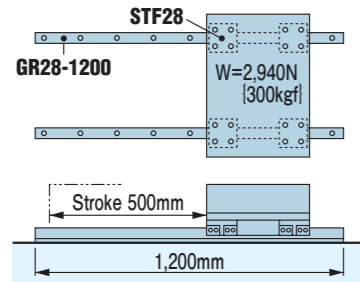


## Durability Test Data / To Prevent Malfunctioning

### Durability Test Data

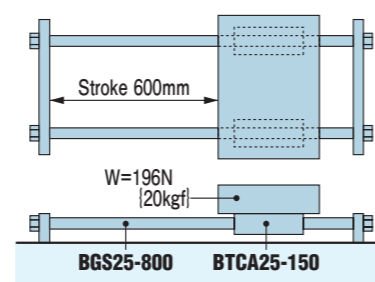
#### S Type

<Testing conditions>	<Result>
Type: STF28 four shift tables GR28-1200 dual-axis	Wear amount on liner: 0.025mm on rail: 0.005mm
Load: 2,940N {300kgf}	Coefficient of friction: 0.08~0.14
Velocity: 0.33m/s {20m/min}	Temperature of friction: 32~42°C
Stroke: 500mm	
Sliding distance: 1,000km	



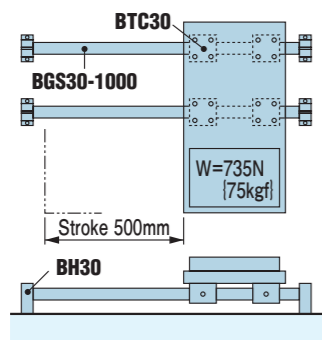
#### BA Type

<Testing conditions>	<Result>
Type: BTCA25-150 one shift table BGS25-800 dual-axis	Wear amount on bushing: 0.055mm on shaft: 0.008mm
Load: 196N {20kgf}	Coefficient of friction: 0.20~0.28
Velocity: 0.50m/s {30m/min}	
Stroke: 600mm	
Sliding distance: 1,000km	



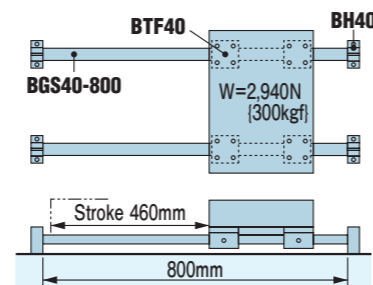
#### BC Type

<Testing conditions>	<Result>
Type: BTC30 four shift tables BGS30-1000 dual-axis	Wear amount on bushing: 0.032mm on shaft: 0.006mm
Load: 735N {75kgf}	Coefficient of friction: 0.12~0.30
Moment: 323N·m {33kgf·m}	
Velocity: 0.25m/s {15m/min}	
Stroke: 500mm	
Sliding distance: 300km (300,000 cycles)	



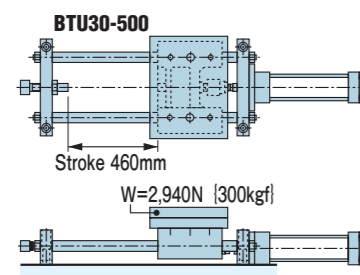
#### BF Type

<Testing conditions>	<Result>
Type: BTF40 four shift tables BGS40-800 dual-axis	Wear amount on bushing: 0.035mm on shaft: 0.008mm
Load: 2,940N {300kgf}	Coefficient of friction: 0.10~0.25
Velocity: 0.42m/s {25m/min}	Temperature of friction: 42~85°C
Stroke: 460mm	
Sliding distance: 1,000km	



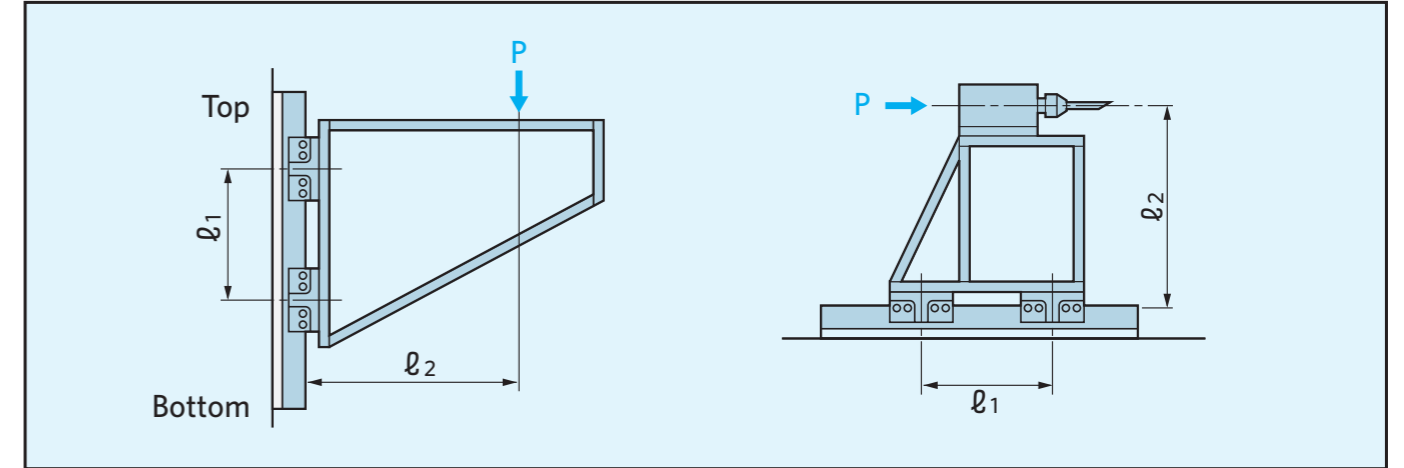
#### BTU Type

<Testing conditions>	<Result>
Type: BTU30-500	Wear amount on bushing: 0.023mm on shaft: 0.012mm
Load: 2,940N {300kgf}	Coefficient of friction: 0.16~0.20
Velocity: 0.42m/s {25m/min}	
Stroke: 460mm	
Sliding distance: 730 (800,000 cycles)	



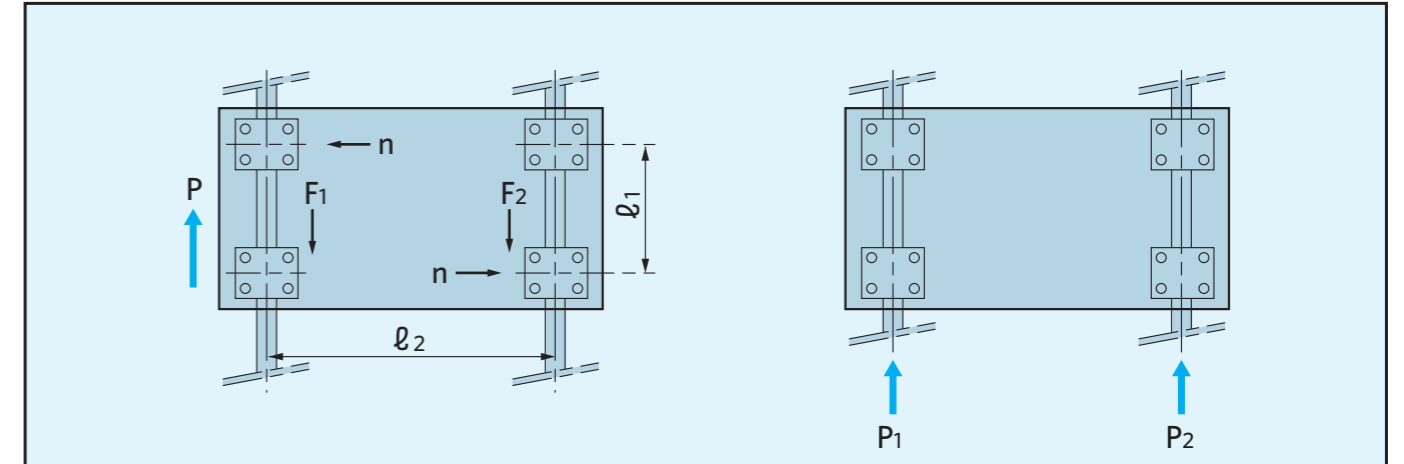
### To Prevent Malfunctioning

- If the point of the drive source is apart from the shift tables



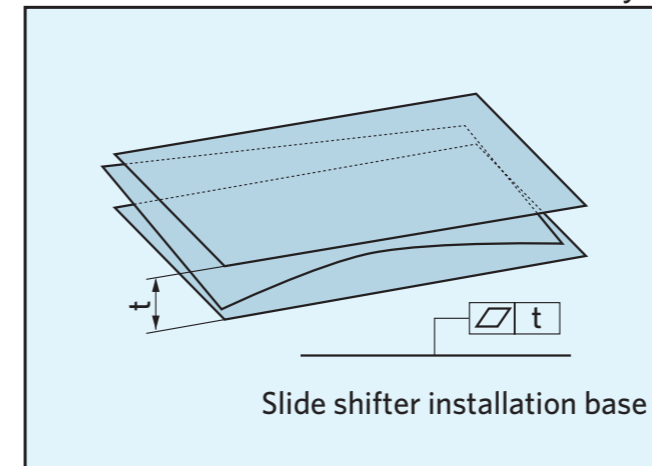
If the position of drive source P is apart from the rail surface by  $l_2$ , of  $l_2/l_1$  exceeds 1.67 when the coefficient of friction  $\mu$  is 0.3, resulting in malfunctioning. Take the allowable moment load into consideration and reduce  $l_2/l_1$  below 1.5.

- If the shift table installation position is apart or the point of the drive source is apart

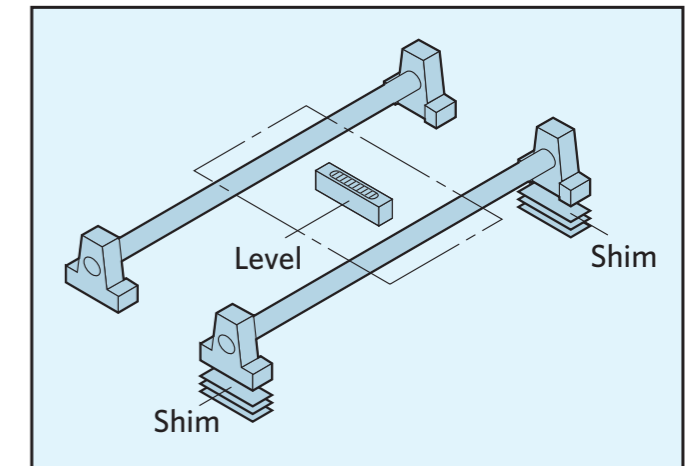


If the  $l_2/l_1$  ratio of dual-axis parallel rails is large, the couple of the drive source P and resistance  $F_1$  and  $F_2$  becomes large and the slide shifter works improperly. Reduce  $l_2/l_1$  below 3. As the point of the drive source becomes apart from the center, the condition becomes worse. Synchronize the drive source with  $P_1$  and  $P_2$  if  $l_2/l_1$  is inevitably larger than 3 for reasons of the structure.

- If the installation base has low accuracy



Do not select the S type if the parallelism  $t$  exceeds 0.3.



Select the B type if the parallelism  $t$  exceeds 0.3. Insert shims under the shaft holders to adjust them. After adjustment, check the parallelism with a level, straight edge, clearance gauge, etc.

# Setting Driving Force And Calculating Service Life

## Setting Driving Force

The driving forces differ with the shift table installation conditions and types of the slide shifters. The typical load conditions and examples of calculating the driving forces are shown below. Take note of them. When setting the driving force, find the load applied to each table first using the formula shown below, and select a proper table in the allowable load table of each series. Then, calculate the driving force from the found load using the formula shown below.

Formula for calculating loads and driving forces

Application form		
	<p>■ Four erecting dual-axis tables</p> $A = W/4 - W/2 \times L_3/L_1$ $B = W/4 - W/2 \times L_3/L_1$ $C = W/4 + W/2 \times L_3/L_1$ $D = W/4 + W/2 \times L_3/L_1$	<p>■ Four erecting dual-axis tables</p> $A = W/4 + W/2 \times L_2/L_0 - W/2 \times L_3/L_1$ $B = W/4 - W/2 \times L_2/L_0 - W/2 \times L_3/L_1$ $C = W/4 - W/2 \times L_2/L_0 + W/2 \times L_3/L_1$ $D = W/4 + W/2 \times L_2/L_0 + W/2 \times L_3/L_1$
	<p>Load calculation Formula</p> $A = W/4 - W/2 \times L_3/L_1$ $B = W/4 - W/2 \times L_3/L_1$ $C = W/4 + W/2 \times L_3/L_1$ $D = W/4 + W/2 \times L_3/L_1$	
Driving force calculation Formula	<p>Driving force = <math>F</math>, safety ratio = <math>S</math> (standard value: 1.5 to 2.5)</p> $F = (\mu_1 A  + \mu_2 B  + \mu_3 C  + \mu_4 D  + nFs) \times S$ <p>The obtained value is a practical driving force. This formula applies to the case where the driving force position is at the table center.</p>	

Application form		
	<p>■ Four lengthwise vertical dual-axis tables</p> $A = B = C = D = W/2 \times L_2/L_0$ $E = F = G = H = W/2 \times L_3/L_1$	<p>■ Four lengthwise horizontal dual-axis tables</p> $A = B = C = D = W/2 \times L_3/L_1$ $E = H = W/4 + W/2 \times L_2/L_0$ $F = G = -W/4 + W/2 \times L_2/L_0$
	<p>Load calculation Formula</p> $A = B = C = D = W/2 \times L_2/L_0$ $E = F = G = H = W/2 \times L_3/L_1$	
Driving force calculation Formula	<p>Driving force = <math>F</math>, safety ratio = <math>S</math> (standard value: 1.5 to 2.5)</p> $F = (\mu_1 A  + \mu_2 B  + \mu_3 C  + \mu_4 D  + nFs + (*nMs) + \mu_1 E  + \mu_2 F  + \mu_3 G  + \mu_4 H ) \times S$ <p>Add the (*nMs) value when the metal scraper is used.</p> <p>The obtained value is a practical driving force. <math>\pm W</math> (weight) is added to the value of the lengthwise vertical tables. This formula applies to the case where the driving force position is at the table center.</p>	

A to H: Load applied to each table  
 n: Number of tables  
 Fs: Seal friction (S Type)/Operating resistance (SE Type)  
 Ms: Metal scraper friction  
 See the description for the values of Fs and Ms.

$\mu$  = Coefficient of friction ( $\mu_1$  and  $\mu_2$  are coefficient of frictions of the tables.)

Type	S Type			Others
Installation condition	Erecting	Lateral	Hanging	
Coefficient of friction $\mu$	0.15	0.17	0.30	0.15

※ Do not use the STC20 or SE type in the lateral or hanging condition.

## Calculating Service Life

### CAUTION

All Oiles slide shifter series employ Oiles bearings and work without the need for lubrication, in principle. If they are lubricated, foreign matter may be removed and the durability is improved further. The service life of the Oiles slide shifters depends on the speed, environmental conditions, etc. Take the condition shown below into consideration and find the service life using the service life calculation formula. It will be the basis of design.

Service life calculation formula  
 (N: Number of durability test cycles)

$$N = a \times K \times \frac{1}{2S} \times \frac{Wa}{Wi} \times \frac{fc}{fw \times fv \times fe \times fl}$$

- $a$  = allowable wear amount [mm]  
 Determine the amount of wear allowed to ensure the accuracy in the machine in use.  
 \* BTCA and BTSa: 0.15 mm or less
- $K$  = coefficient of friction  
 The slide shifters use various types of Oiles bearings. The coefficient of friction is set according to their performances and using conditions.

Series	BTCA and BTSa	Other series
Coefficient of friction $K$	$1.2 \times 10^7$	$5 \times 10^6$

- $S$  = stroke [m]
- $Wa$  = allowable load [N {kgf}]  
 See the Parts List.
- $Wi$  = load [N {kgf}]  
 Select the severest one (i.e., value nearest the allowable load) at the load applied to each table calculated with the load calculating formula.

- $fc$  = contact factor  
 When several shift tables are used, it is difficult to obtain uniform load distribution due to influences of the installation surface accuracy. This factor is determined by taking the contact conditions of the shift tables into consideration.

Number of tables per axis	One	Two	Three	Four
Contact factor $fc$	1.0	0.75 to 0.85	0.65 to 0.75	0.6 to 0.65

- $fw$  = load factor  
 This factor is used for compensation of the load conditions, since it is difficult to grasp the actual vibrations, impacts, etc. produced in machines.

Impact fluctuating loads	None	Yes
Load factor $fw$	1.0	2.0 to 3.0

- $fv$  = velocity factor  
 This factor is used for compensation of the speed conditions.

Velocity condition	0.0017 or less	0.0017 to 0.05	0.05 to 0.5	0.5 to 1
Velocity factor $fv$	0.25 to 0.3	0.3 to 1.0	1.0 to 2.0	2.0 to 4.0

(Unit: m/sec.)

- $fe$  = environmental factor  
 This factor is used for compensation of the ambient temperature and influences of foreign matter.

Ambient temperature	60°C or less		60 to 100°C	
Foreign matter	None	Yes	None	Yes
Environmental factor $fe$	1.0 to 2.0	3.0 to 6.0	3.0 to 6.0	6.0 to 12.0

※ Lubrication is needed if the temperature is 100°C or more.

- $fl$  = lubrication factor  
 Lubrication enhances the effect in high-speed operation or foreign matter need to be removed.

Running distance and lubrication	No lubrication	Lubrication every 10 km	Lubrication every 1 km
Lubrication factor $fl$	1.0 to 1.5	0.3 to 0.5	0.2 to 0.3

※ Use the BTCA and BTSa in the self-lubricating conditions.

# Calculation Examples of Four Erecting Dual-Axis Tables (S Type)

## Load Calculation, Driving Force Calculation and Service Life Calculation

Use conditions and required service life

N=1 million cycles W=100kgf       $f_c = 0.75$ , single-axis, two units       $f_e = 1.0$ , 60°C or less, no foreign matter  
 $a = 0.1\text{mm}$        $K = 5 \times 10^6$        $f_w = 1.0$ , no impact load       $f_l = 1.0$ , no lubrication  
 $S = 0.25\text{m}$        $L_0 = L_1 = L_2 = L_3 = 200\text{mm}$        $f_v = 1.0$ , 0.05m/s

### ① Calculating loads applied to tables

$$A = W/4 + W/2 \times L_2/L_0 - W/2 \times L_3/L_1$$

$$B = W/4 - W/2 \times L_2/L_0 - W/2 \times L_3/L_1$$

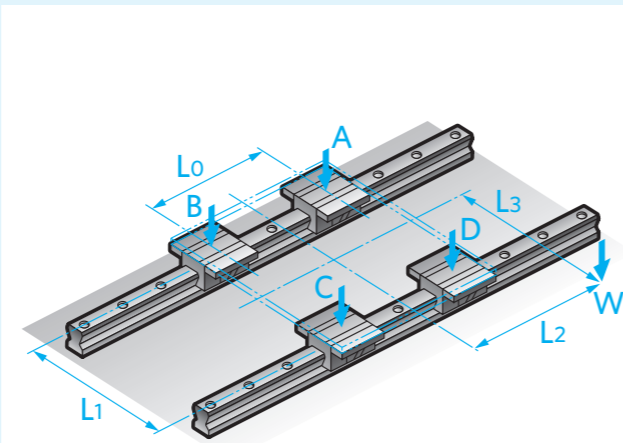
$$C = W/4 - W/2 \times L_2/L_0 + W/2 \times L_3/L_1$$

$$D = W/4 + W/2 \times L_2/L_0 + W/2 \times L_3/L_1$$

Use the load calculating formula shown above and find the load to be applied to each table from the use conditions.

W = load = 100 kgf  
 $L_0, L_1, L_2, L_3$  = distance between load application point and each table  
 Find A, B, C and D (load to be applied to each table) from  $L_0 = L_1 = L_2 = L_3 = 200\text{mm}$ .  
 $A = 100/4 + 100/2 \times 200/200 - 100/2 \times 200/200 = 25$   
 $B = 100/4 - 100/2 \times 200/200 - 100/2 \times 200/200 = -75$   
 $C = 100/4 - 100/2 \times 200/200 + 100/2 \times 200/200 = 25$   
 $D = 100/4 + 100/2 \times 200/200 + 100/2 \times 200/200 = 125$

Select the proper shift tables so that the values of A, B, C and D are within the ranges of values in the allowable load table shown in the description.  
 A = 25 kg (erecting)  
 B = -75 kg (hanging) ※See the hanging values for negative values.  
 C = 25 kg (erecting)  
 D = 125 kg (erecting)  
 As a result, the STF28 is selected from the allowable load table.



### Allowable load with simple load

(Unit: N {kgf})

Part No.	Load type	Installation condition		
		Erecting	Lateral	Hanging
STC20	Allowable static load	10,800 { 1,100}	—	—
	Allowable dynamic load	3,430 { 350}	—	—
STC28 · STF28	Allowable static load	17,700 { 1,800}	4,410 { 450}	3,920 { 400}
	Allowable dynamic load	5,880 { 600}	1,470 { 150}	1,270 { 130}
STF38	Allowable static load	31,400 { 3,200}	7,850 { 800}	7,360 { 750}
	Allowable dynamic load	10,800 { 1,100}	2,650 { 270}	2,450 { 250}
STF48	Allowable static load	44,100 { 4,500}	13,700 { 1,400}	11,800 { 1,200}
	Allowable dynamic load	14,700 { 1,500}	4,410 { 450}	3,920 { 400}

- Allowable static load: Allowable load when it is born in the stationary condition or at quite low speed near stopping (not more than 0.0017 m/s [0.1 m/min].)
- Allowable dynamic load: Allowable load in the condition with sliding speed of 1.0 m/s [60 m/min] or less.
- Do not use the STC20 in the lateral or hanging condition.

### ② Calculating driving force

$$F = (\mu_1|A| + \mu_2|B| + \mu_3|C| + \mu_4|D| + nF_s + nM_s) \times S$$

Find the driving force from the use conditions using the above formula.

Type	S Type		
	Erecting	Lateral	Hanging
Installation condition	0.15	0.17	0.30
Coefficient of friction $\mu$	0.15	0.17	0.30

※Do not use the STC20 in the lateral or hanging condition.

F = driving force  
 S = safety factor (2 in this example)  
 $\mu^1, \mu^2, \mu^3, \mu^4$  = coefficient of friction of each table  
 $\mu^1 = 0.15$  (erecting)  
 $\mu^2 = 0.30$  (hanging)  
 $\mu^3 = 0.15$  (erecting)  
 $\mu^4 = 0.15$  (erecting)

A, B, C and D = load applied to each table

A = 25 kgf  
 B = -75 kgf  
 C = 25 kgf  
 D = 125 kgf  
 n = number of tables = 4  
 $F_s$  = seal friction = 1.2 kgf

The value of F is obtained as shown below:

$$F = (0.15 \times 25 + 0.3 \times 75 + 0.15 \times 25 + 0.15 \times 125 + 4 \times 1.2) \times 2 \approx 107 \text{ kgf}$$

### ③ Calculating service life

$$N = a \times K \times (1/2S) \times (W_a/W_i) \times f_c / (f_w \times f_v \times f_e \times f_l)$$

Find the service life from the use conditions using the above formula.

N = service life = required service life (1 million cycles)  
 a = allowable wear amount = 0.1 mm  
 K = coefficient of friction =  $5 \times 10^6$   
 S = stroke = 0.25 m  
 $W_a$  = allowable load = 130 kgf (Hanging STF28)  
 $W_i$  = applied load = 75 kgf (Value closest to the allowable load: Select the severest one among the loads found with the load calculating formula.)  
 $f_c$  = contact factor = 0.75 (Two single-axis units)  
 $f_w$  = load factor = 1.0 (No impact load)  
 $f_v$  = velocity factor = 1.0 (0.05 m/s)  
 $f_e$  = environmental factor = 1.0 (60°C or less, no foreign matter)  
 $f_l$  = lubrication factor = 1.0 (No lubrication)

The value of N is found as shown below:

$$N = 0.1 \times 5 \times 10^6 \times 1 \times (2 \times 0.25) \times (130/75) \times 0.75 / (1.0 \times 1.0 \times 1.0 \times 1.0) = 1,300,000$$

The required service life of 1 million cycles is satisfied.